LETTER TO THE EDITORS

The static and dynamic response of a hotwire anemometer

T. W. Davies*

In a recent article, Bullock *et al*¹ presented some predictions of the static and dynamic responses of a hot-wire anemometer. Their predictions were claimed to substantiate the validity of King's law:

$$
E^2 = a + bU^n \tag{1}
$$

where E is the wire voltage corresponding to the effective cooling velocity U , and a, b and n are constants. A model of the dynamic behaviour of a hot-wire anemometer was then developed using Eq (1), leading to the conclusion that a more sophisticated anemometer control system was needed for measurements of large amplitude, high frequency turbulence.

It has been demonstrated by Siddall and Davies² and by Fand and Keswani³ that the static response of a hot-wire anemometer is well represented by an equation of the form:

Fig I Static response of a hot-wire anemometer

Fig 2 Dynamic response of a hot-wire anemometer

$$
E^2 = A + B U^{1/2} + C U \tag{2}
$$

Fig 1 shows the comparison of a least-squares fit of Eqs (1) and (2) to the hot-wire calibration data of Davies and Bruun⁴. These data were obtained using a $5~\mu$ m diameter wire and cover the entire Reynolds number range of interest in anemometry. Fig 1 demonstrates the superiority of Eq (2) over Eq (1) . Correlations of the form of Eq (1) may only be used over limited ranges of Reynolds number, ie for low amplitude turbulence measurements.

Perry and Morrison⁵ demonstrated that the dynamic response of a hot-wire anemometer calculated from Eq (1) for any value of n gives poor agreement with the measured dynamic response.

The method of calculating dynamic flow parameters, such as turbulence intensity, described by Schubauer and Klebanoff⁶, involves the use of wire sensitivity, *dE/d U,* obtained by differentiation of the wire static response equation. Thus the sensitivity of wire based on Eq (1) is:

$$
\frac{\mathrm{d}E}{\mathrm{d}U} = \frac{\mathrm{nb}U^{n-1}}{2E} \tag{3}
$$

and that based on Eq (2) **is:**

$$
\frac{\mathrm{d}E}{\mathrm{d}U} = \frac{2C + B U^{-1/2}}{4E} \tag{4}
$$

Fig 2 shows the dynamic response of the wire used by Davies and Bruun⁴, calculated by numerical differentiation of their calibration data, compared with Eqs (3) and (4).

The conclusion to be drawn from Figs 1 and 2 is that the use of Eq (2) leads to more accurate mean flow velocity and turbulence intensity measurements, especially when large amplitude turbulence fluctuations are involved.

References

- 1 Bullock K. J., Ledwieh M. A. and Lai J. C. S. Numerical simulation of transient response of heat transfer from a hot-wire anemometer transducer. *Int. J. Heat and Fluid Flow,* 1985, 6(1), 57-65
- 2 Siddall R. G. and Davies T. W. An improved response equation for hot-wire anemometry. *Int. J. Heat Mass Transfer,* 1971,15, 367-368
- 3 Fand R. M. **and Keswanl** K. K. A continuous correlation equation for heat transfer from cylinders to air in cross flow from Reynolds numbers from 10^{-2} to 2×10^5 , *ibid*, 1971, 15, 559-562
- 4 Davies P. O. A. L. **and Bruun** H. H. The performance of a yawed hotwire. *Proc. Syrup. Instrumentation and Data Processing for Industrial Aerodynamics, NPL,* 1968, 10.1-10.12
- 5 Perry A. E. and Morrison G. L. Static and dynamic calibration of constant temperature hot-wire systems. *J. Fluid Mech.* 1971, 47, 765-777
- Schubauer G. B. and Klebanoff P. S. Theory and application of hotwire anemometers in the investigation of turbulent boundary layers. *NACA ACR No. 5K27,* 1946

^{*} Department of Chemical Engineering, University of Exeter, North Park Road, Exeter, EX4 4QF, UK Received 3 May 1985 and accepted for publication on 3 June 1985